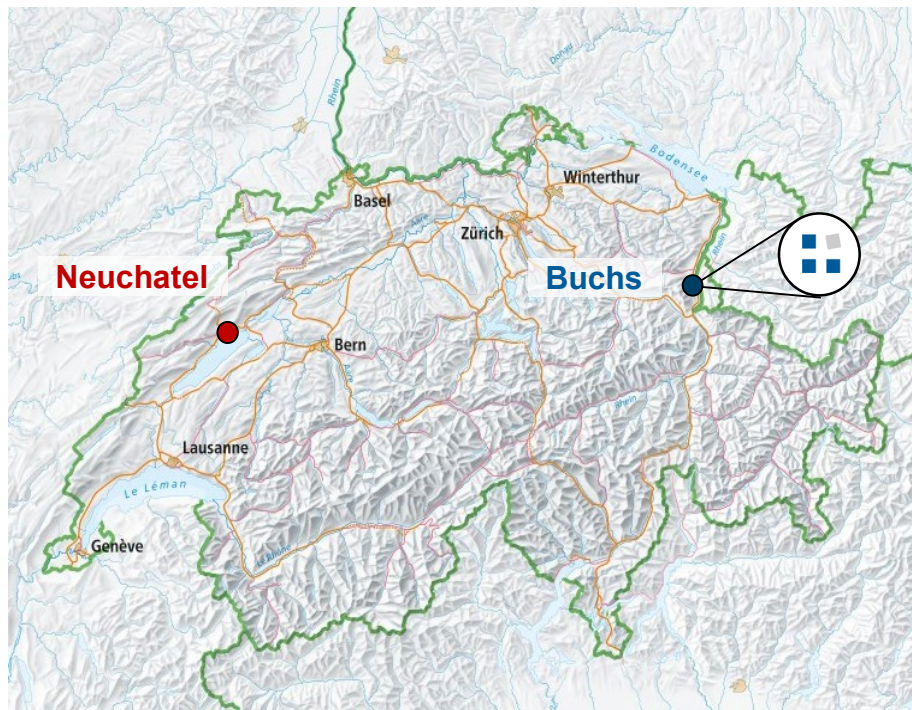


Institute for Micro and Nanotechnology MNT SPPL- Cooling methods for high thermal load devices



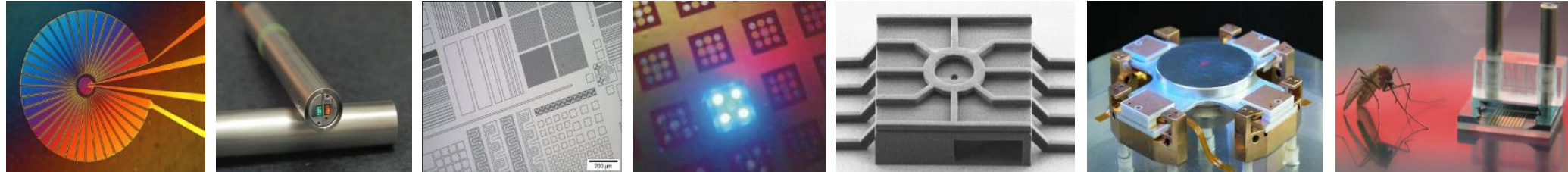
Interstate University of Applied Science Buchs, NTB

- Member of the FHO
- Systems engineering with 6 profiles
- 400 Students, 40 Professors, 150 Staff
- 7 Institutes
- 49 years on market (Nov. 1970)

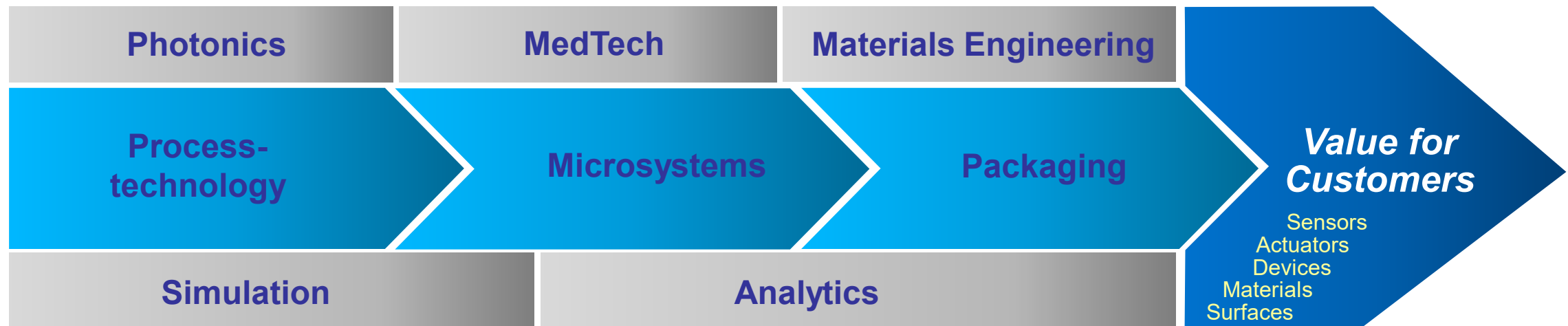
Technology Transfer as «Partner of Industry»

- Services in processing, metrology and consulting
- Applied R&D projects
- Feasibility studies
- Diploma works
- Different hands on courses (MEMS, Vacuum,...)

■ Our core competencies, infrastructure, processes

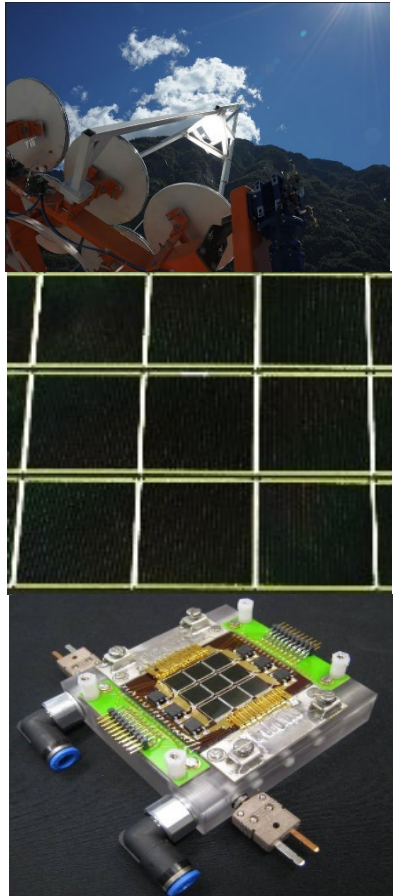


- Sensors and Actuators (MEMS)
- Photonics, Micro-optical Systems (MOEMS)
- Packaging Technologies
- Thin Films, Micro- and Nanostructures
- MedTech & Microfluidics Devices
- Characterization & Failure Analysis
- Polymers and Adhesives
- Additive Manufacturing

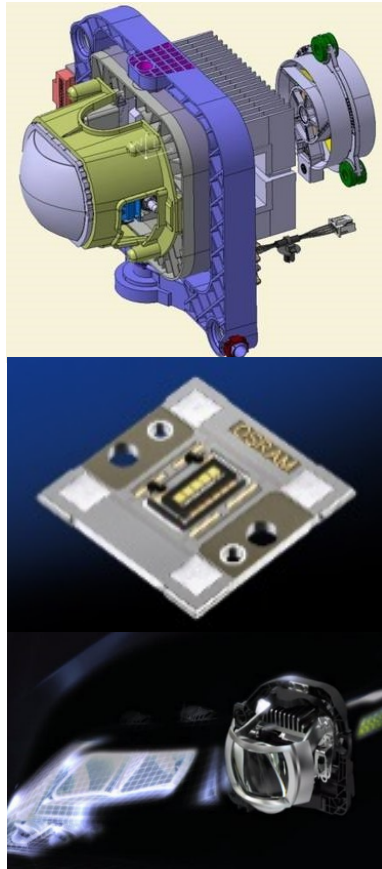


Why thermal management

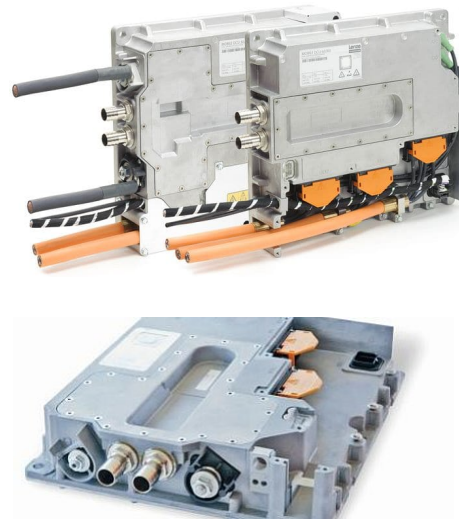
Solar cells, LEDs, power electronics



© IBM, DSolar



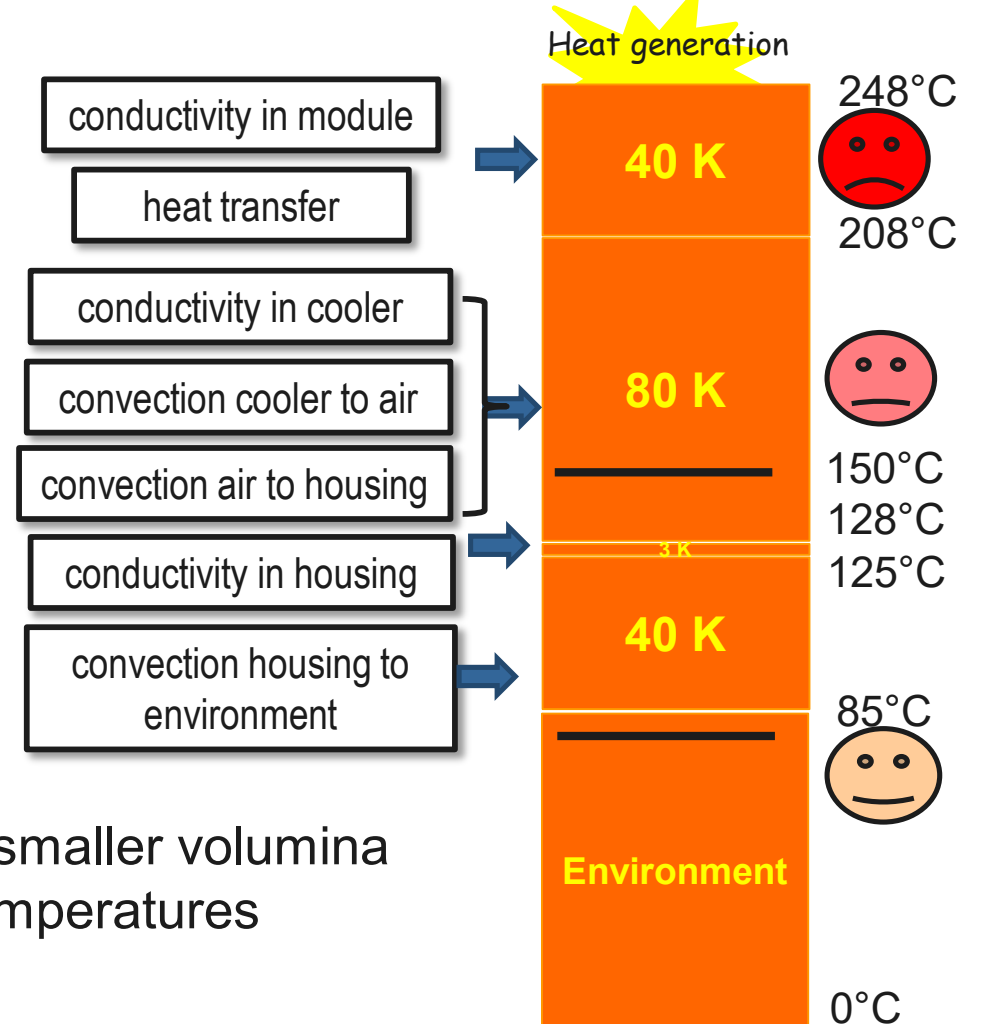
© OSRAM, Hella



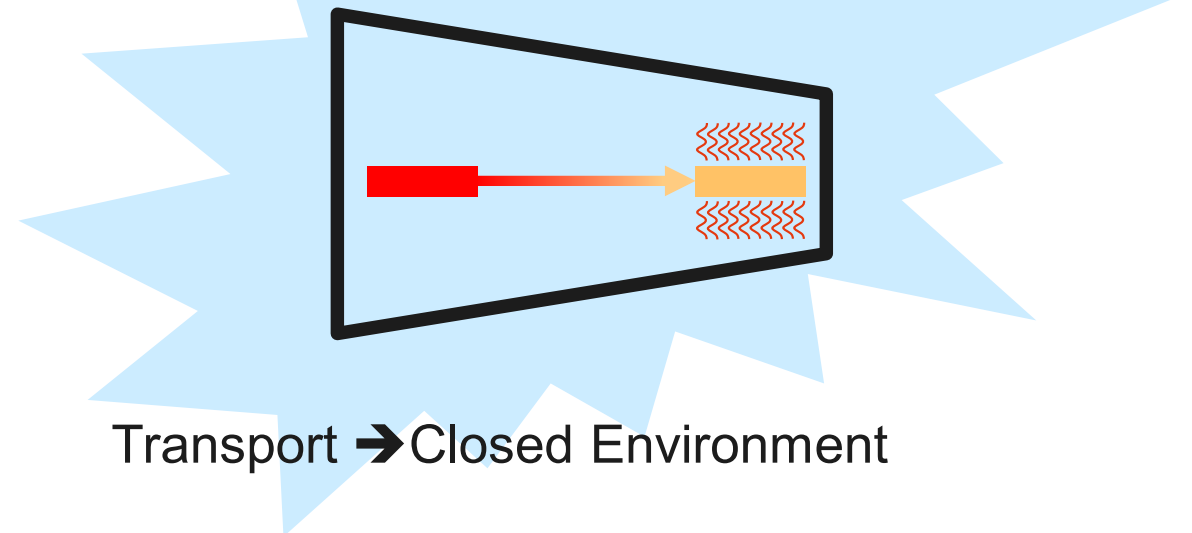
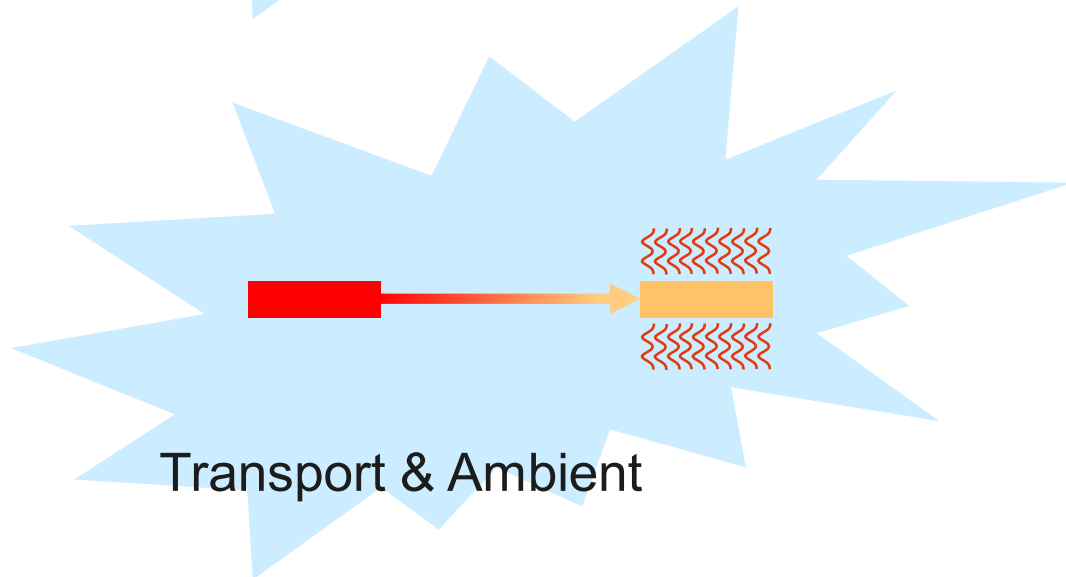
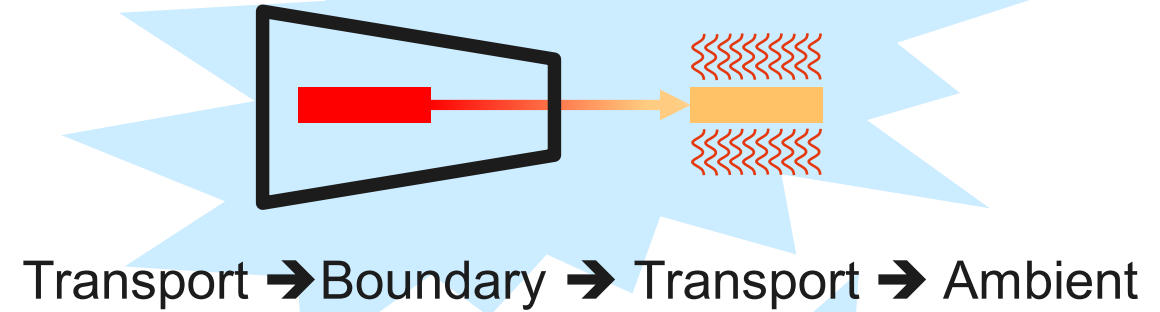
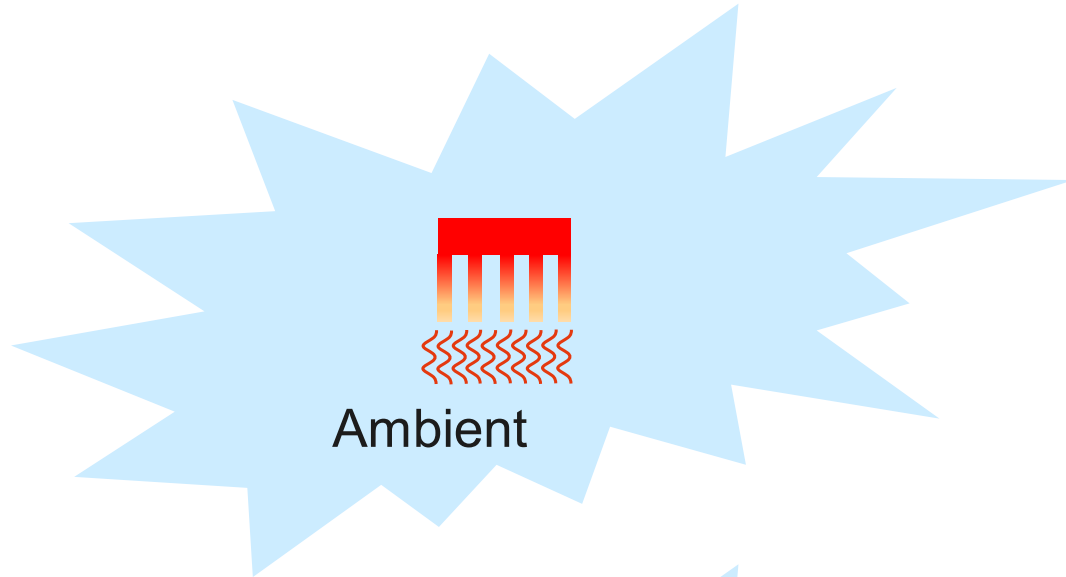
© Lenze Schmidhauser

High thermal loads in smaller volumina and higher ambient temperatures

e.g.: LED in car frontlight



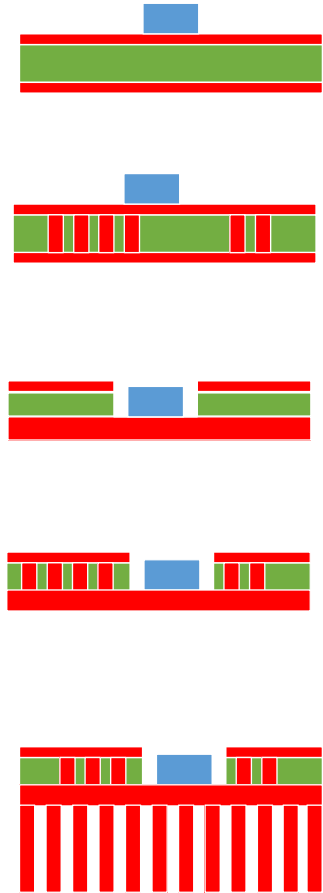
Where is the system boundary (closed loop?)



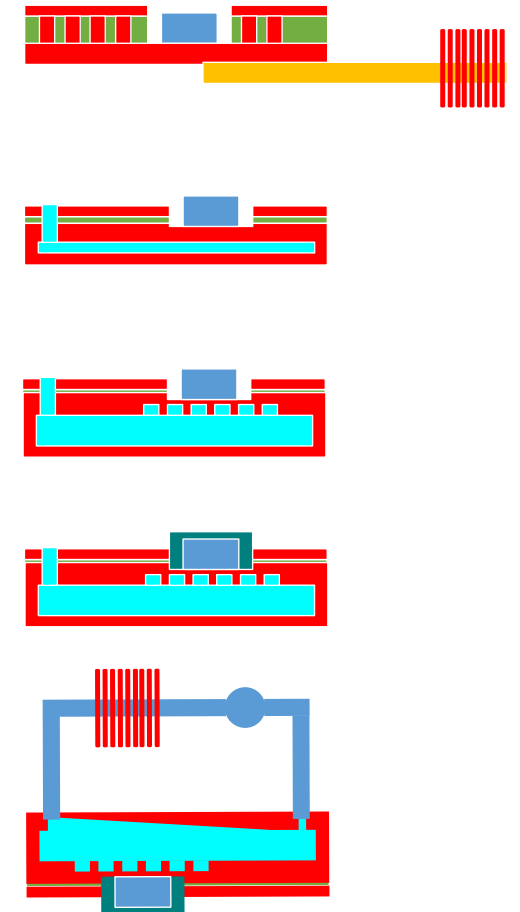
What is done today

- **Smaller** traces and **thicknesses** for the copper and isolators, but no vast improvement in thermal conductivity of the isolators, other than **reducing layers** and base material thicknesses for PCBs.
- Potting of devices using thermal conductive adhesives / Pastes to have better heat transfer and homogenisation in a device (**no unfilled air rooms**)
→ Nowadays projects to improve the thermal conductivity using **particle filled** mould systems.
- **Populating** the Electronic boards onto the «cool» / «**cooled surfaces**»
- **Using the available cooling systems** for example a cars air conditioning to cool the electronic parts.
- Special coatings to have «submerged» electronics

Strategies for cooling

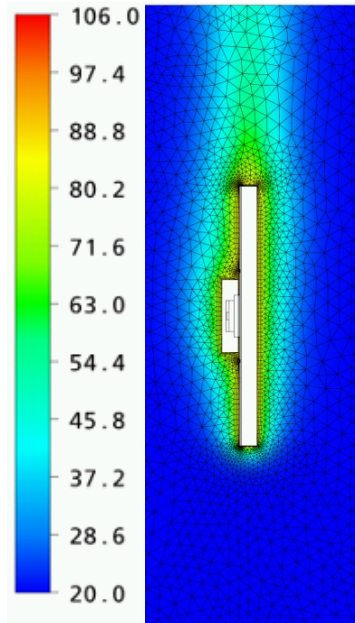
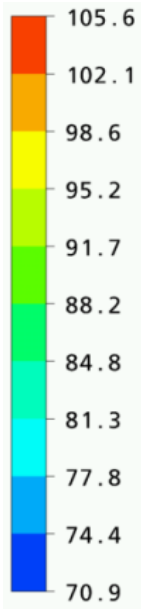
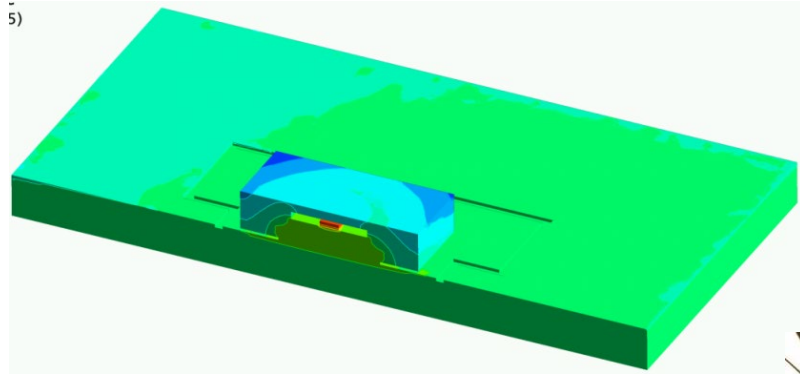


Principle Medium	no external drive	External drive (motor, compressor unit)
1 Phase: Air	natural convection & heat sink	forced convection (heat sink & fan)
1 Phase: Liquid	natural convection using density differences	liquid pump
2 Phase: liquid and vapour	heatpipe vapour bubble pump	refrigerator pump
Short time / Power bursts	phase change materials	Peltier element

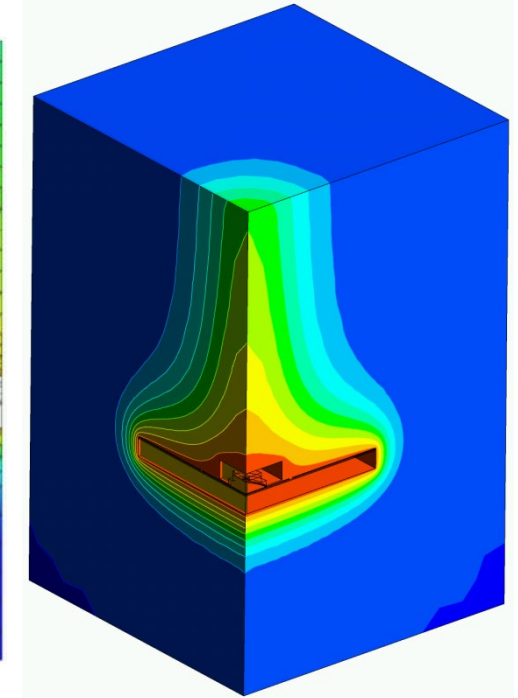
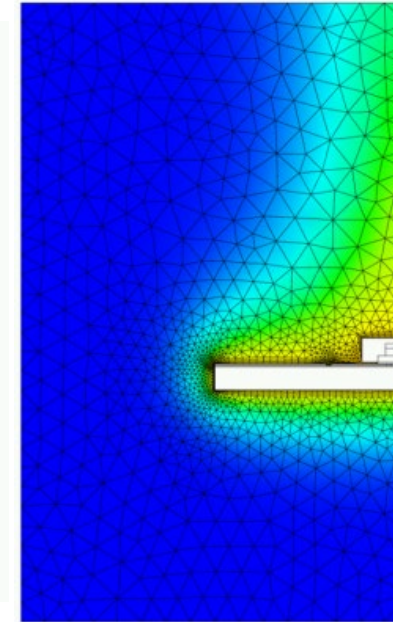
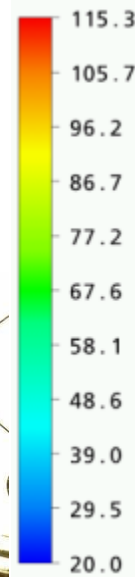
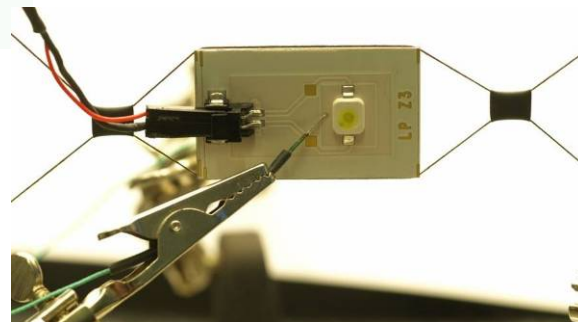


Also mixing of the above mentioned methodes
 (e.g.: liquid pump combined with vapour bubbles) *Fundamentals of Heat and Mass Transfer*
 Incropera, DeWitt

Natural convection



Vertical System

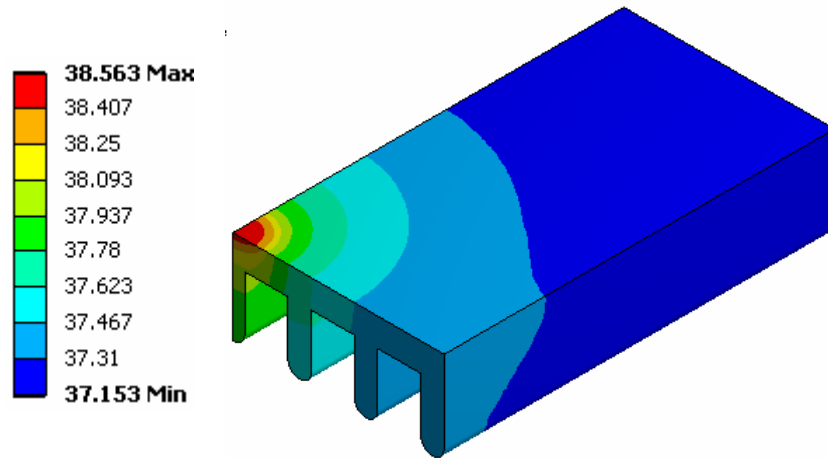


$T_{\infty} = 20 \text{ }^{\circ}\text{C}$
 $T_{\text{Substrate}} = 85.6 \text{ }^{\circ}\text{C}$
 $\alpha = 9.96 \text{ W}/(\text{m}^2 \cdot \text{K})$
 $R_{\text{th BA}} = 65.6 \text{ K/W}$

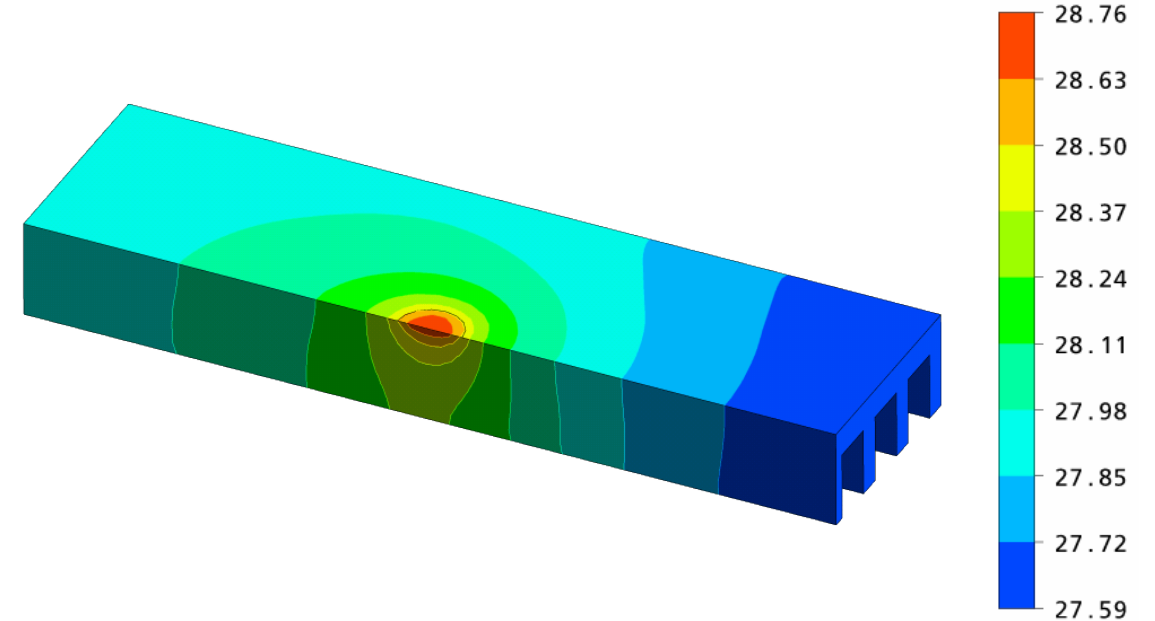
$T_{\infty} = 20 \text{ }^{\circ}\text{C}$
 $T_{\text{Substrate}} = 94.8 \text{ }^{\circ}\text{C}$
 $\alpha = 8.73 \text{ W}/(\text{m}^2 \cdot \text{K})$
 $R_{\text{th BA}} = 74.8 \text{ K/W}$

Horizontal System

Heat sink natural or forced convection

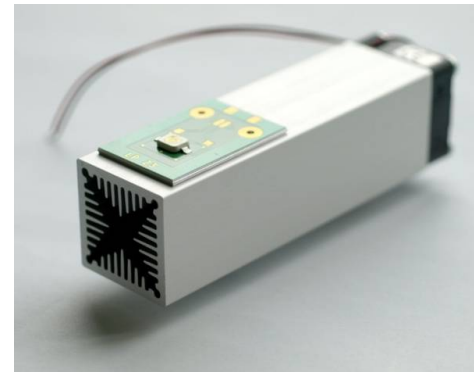
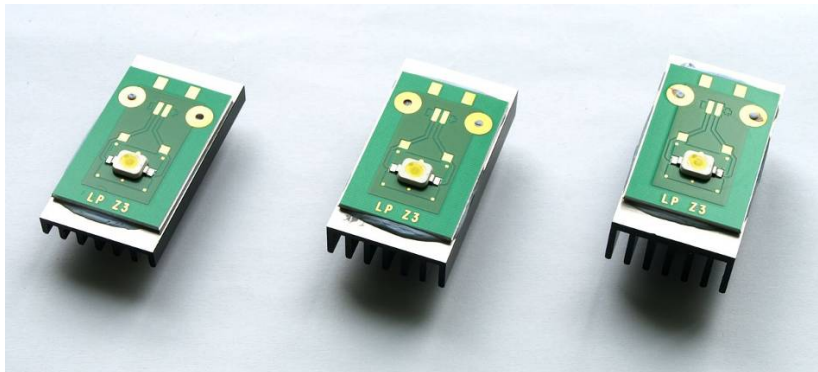


Einspeisung punktförmig
 $R_{th} = 17.9 \text{ K/W}$
 (bezogen auf mittlere Temperatur)



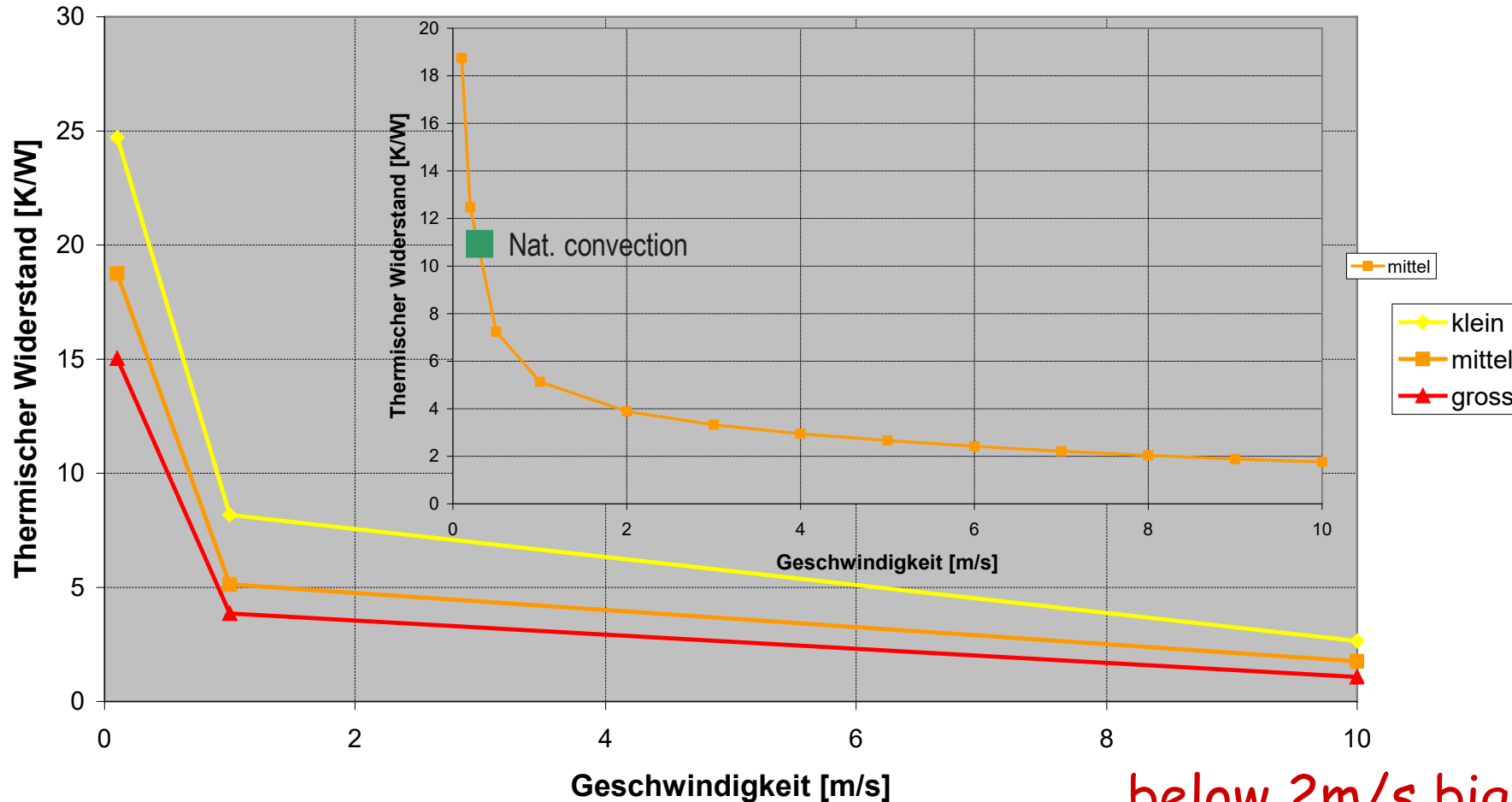
$v_{Luft} = 1 \text{ m/s}$
 $T_{\infty} = 20 \text{ }^{\circ}\text{C}$
 $T_{Substrat} = 28.1 \text{ }^{\circ}\text{C}$
 $\alpha = 23.6 \text{ W}/(\text{m}^2 \cdot \text{K})$

$R_{th \text{ BA}} = 8.1 \text{ K/W}$



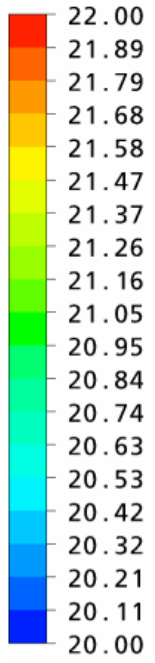
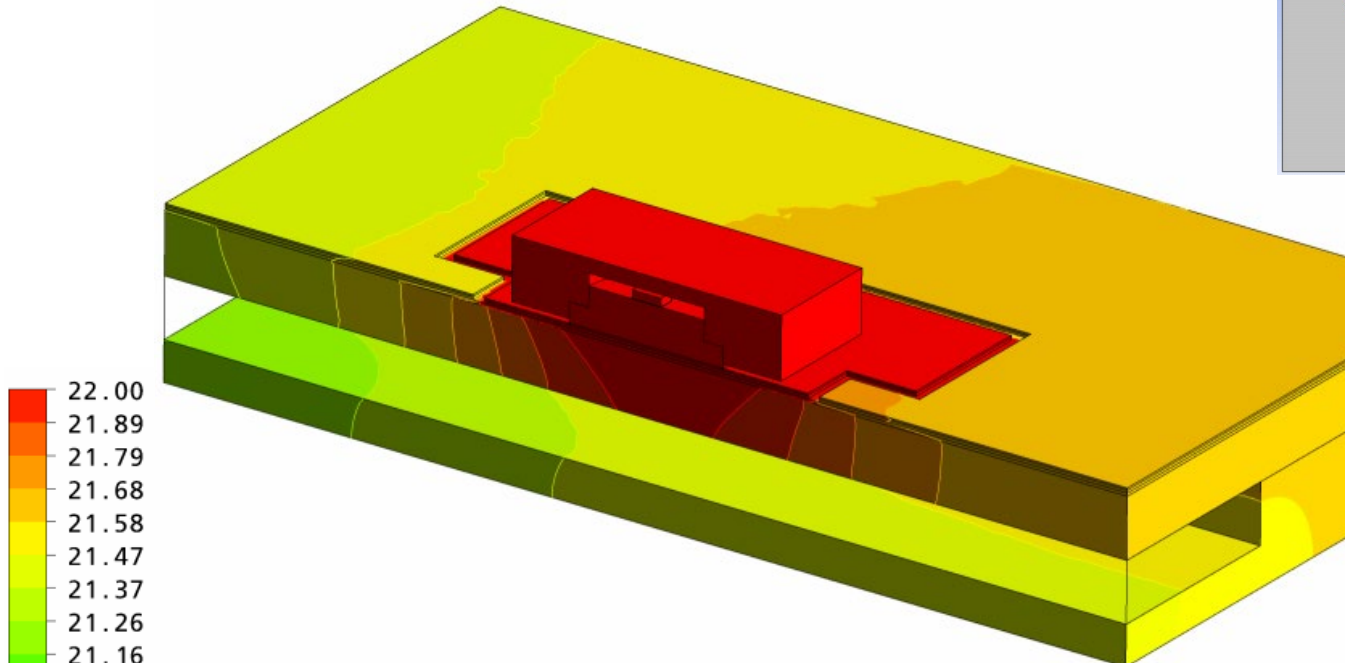
Heat sink and airflow behaviour

Thermischer Widerstand als Funktion der Luftgeschwindigkeit

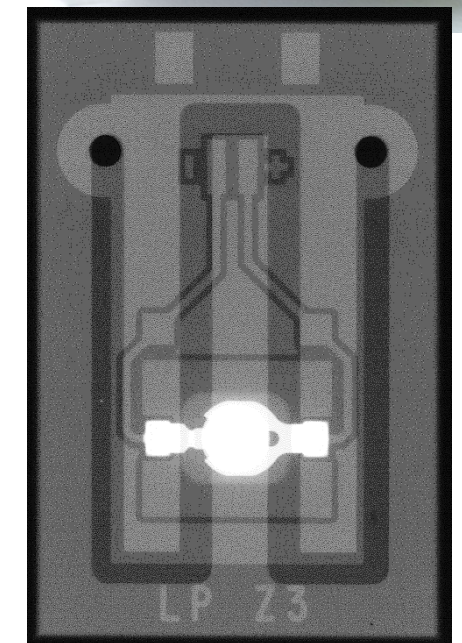
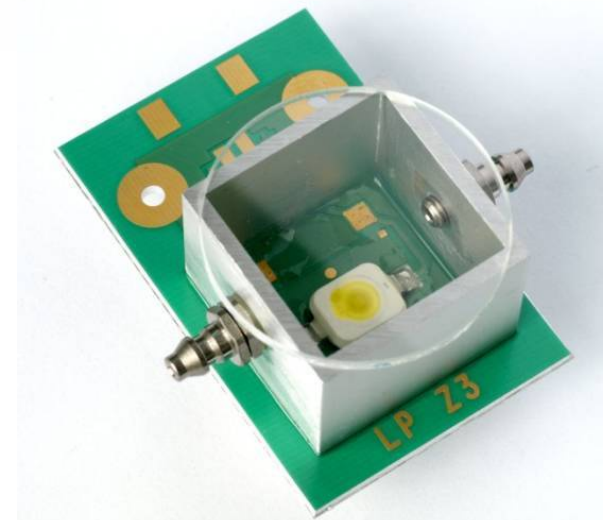
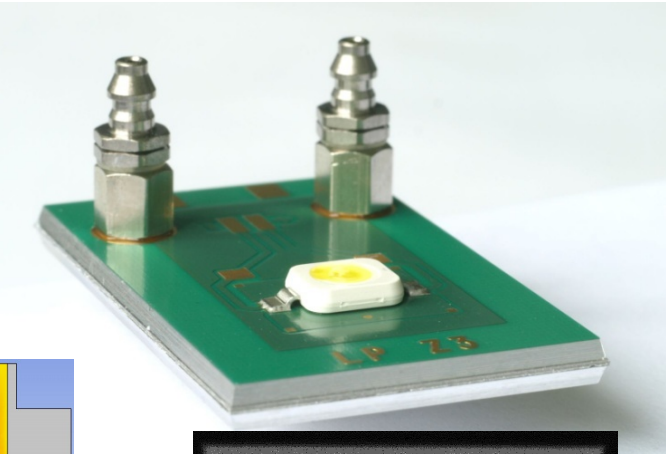
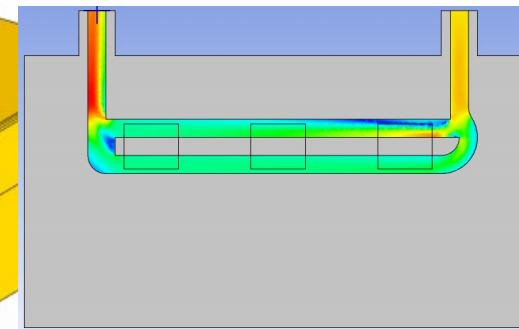
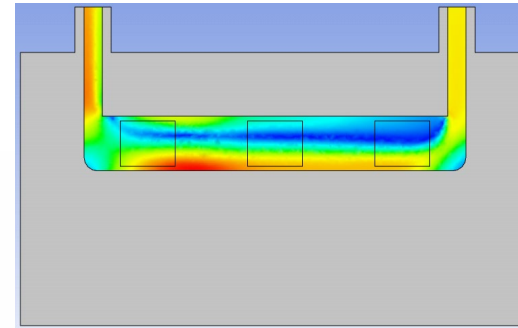


below 2m/s big influence

Flow in IMS Substrate

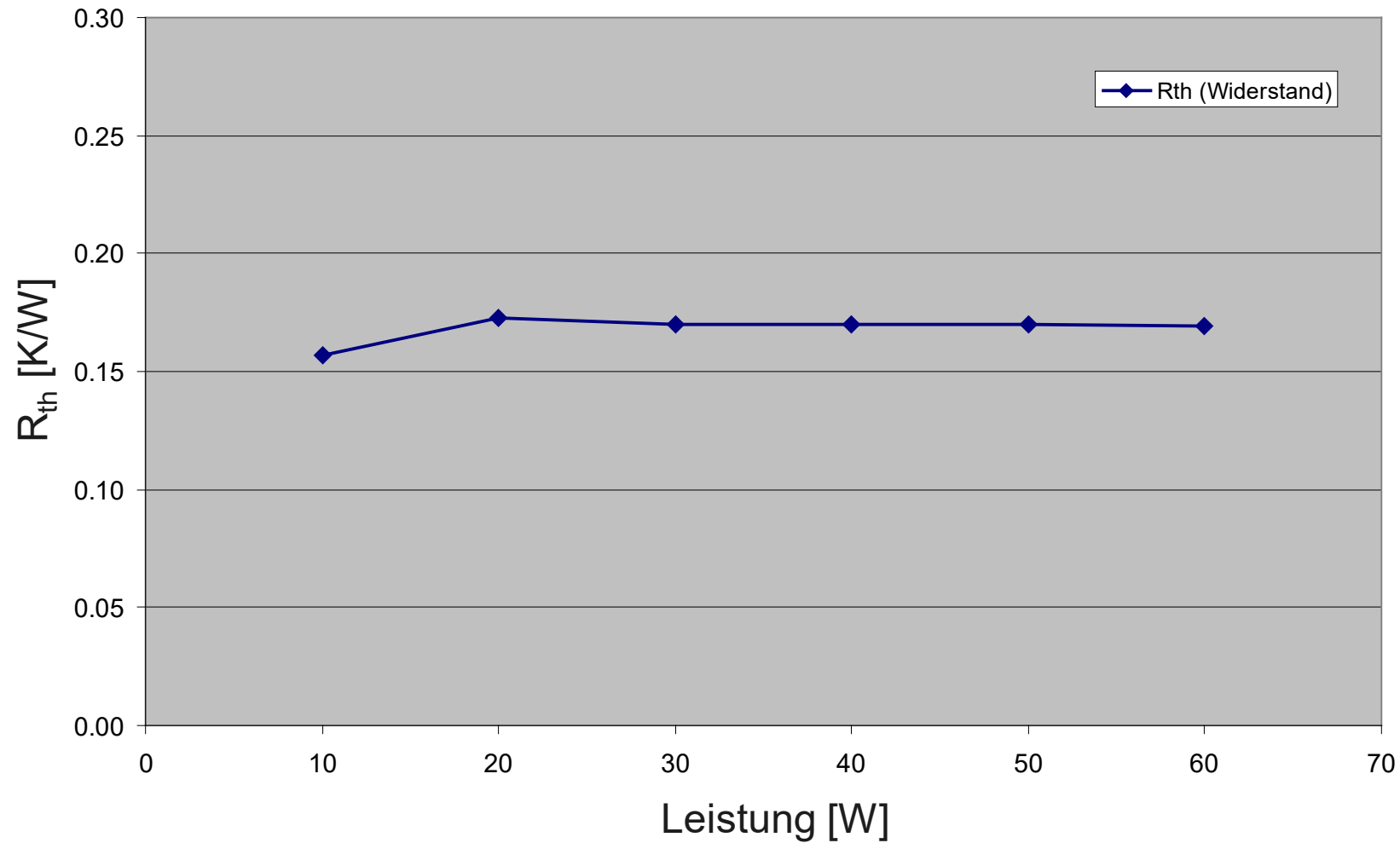


Flow: 1000 ml/h
 $T_{\infty} = 20 \text{ }^{\circ}\text{C}$
 $T_{\text{Substrat}} = 21.6^{\circ}\text{C}$
 $\alpha = 1129 \text{ W}/(\text{m}^2 \cdot \text{K})$
 $R_{\text{th BA}} = 1.6 \text{ K/W to } 0.31 \text{ K/W}$
 Depending on Channel geometry and flow rate

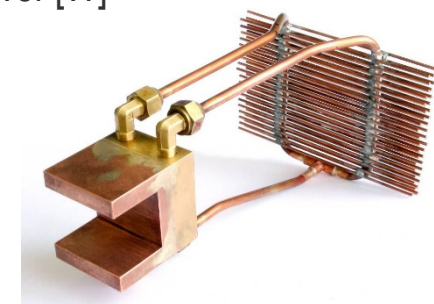
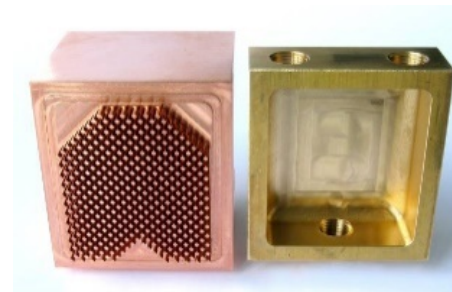
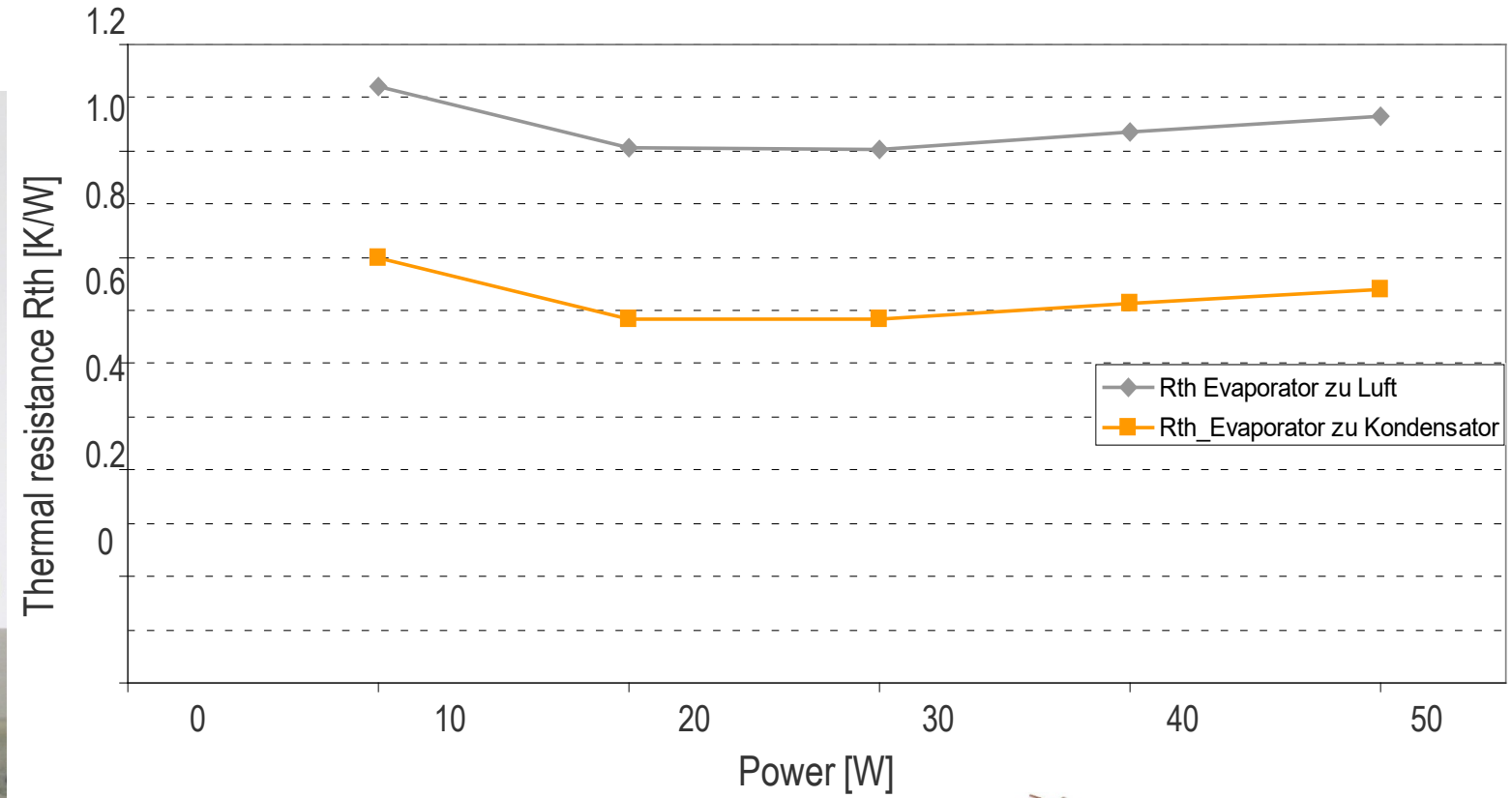
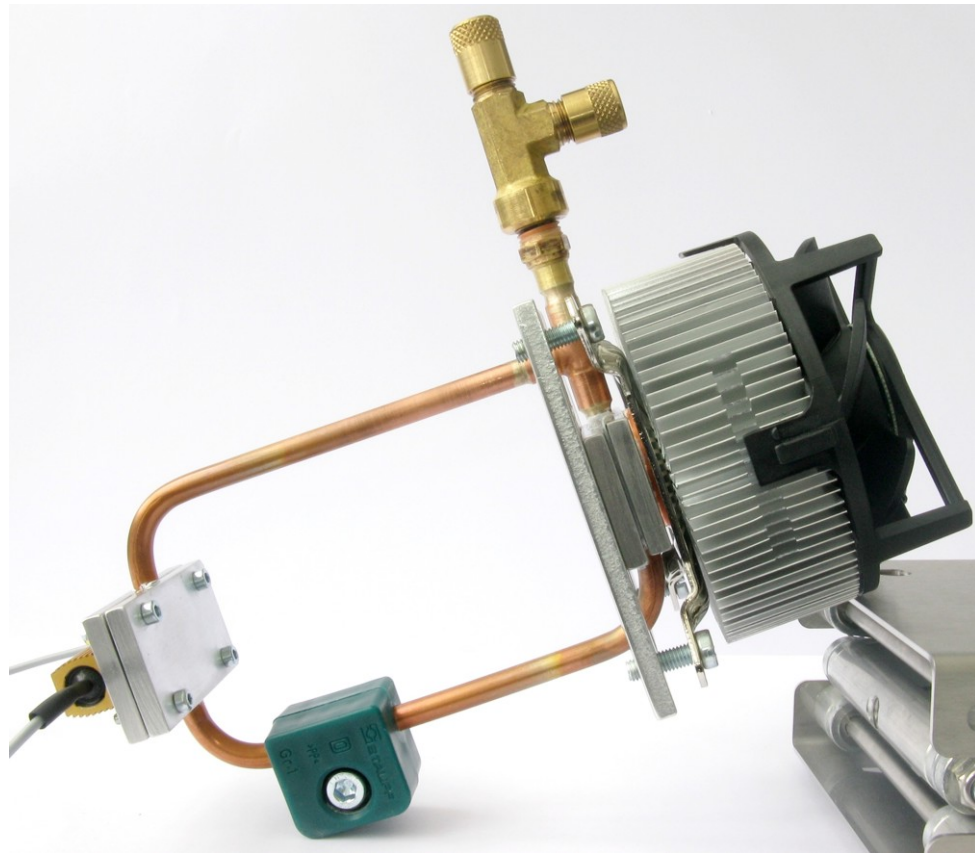


Tested to 28 bar pressure

Thermal resistance of a closed fluid loop system

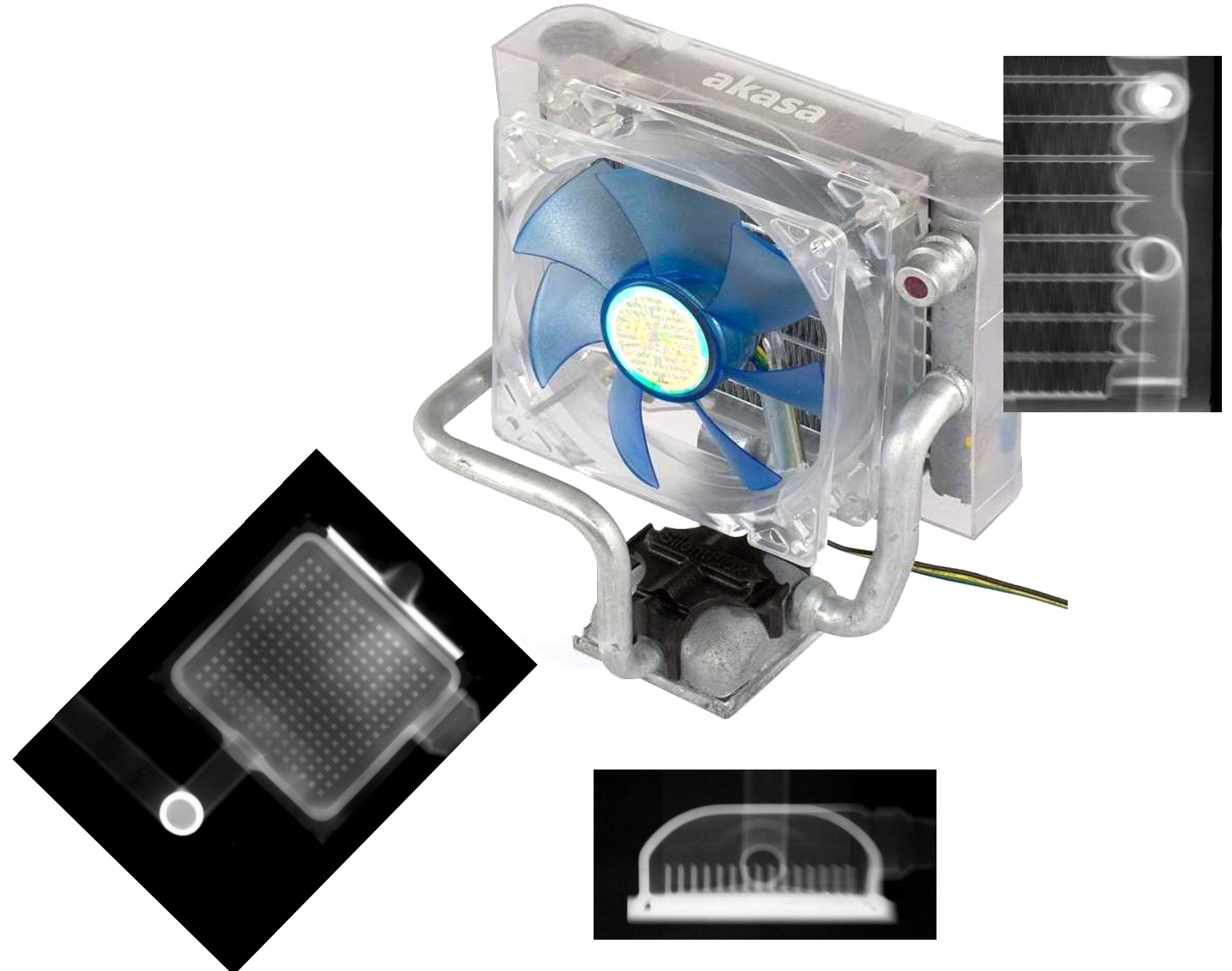


Vapour bubble pump with air cooling

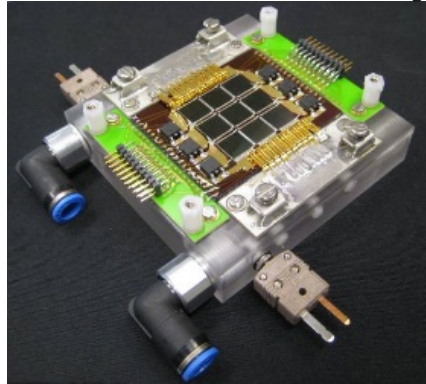


REVO processor cooler using bubble pump

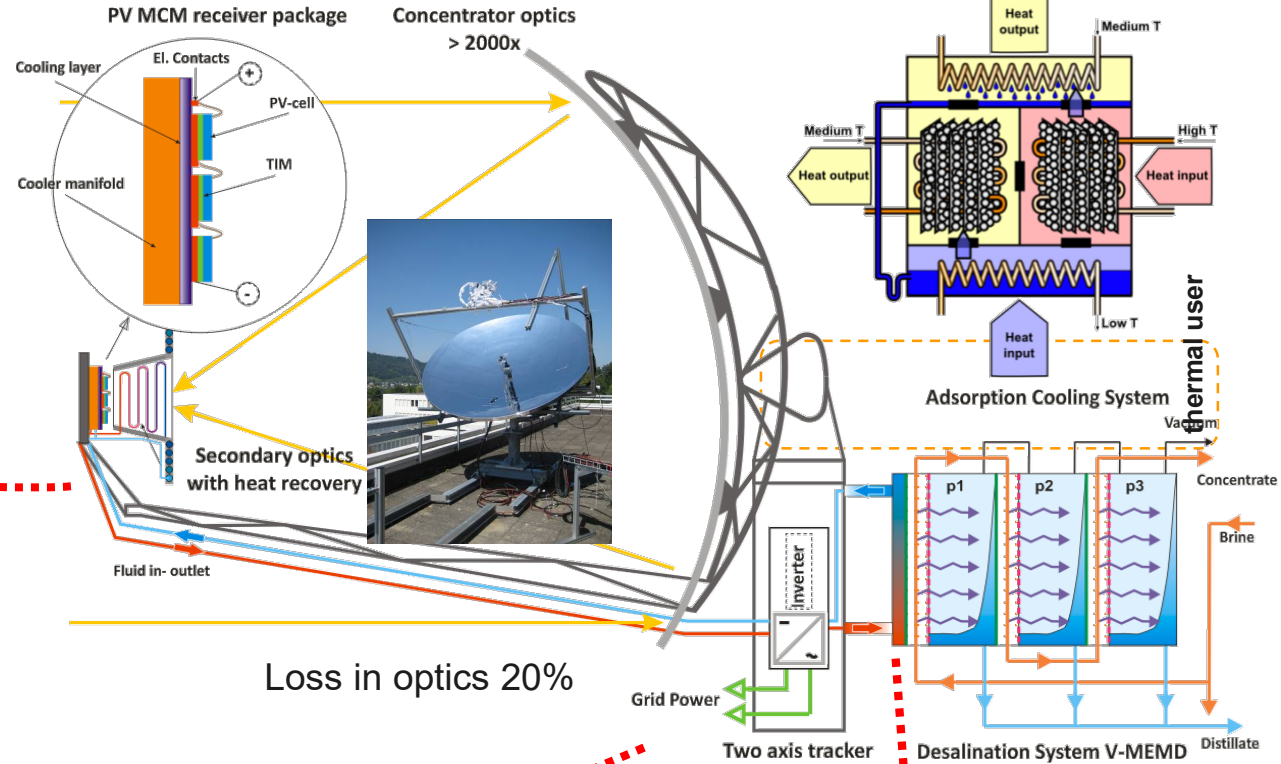
- Closed system loop
- Medium: R134a
- Al-construction
- Fine pitch fin cooler
- Different orientations possible
- $R_{th} = 0.3 \dots 0.4 \text{ K/W}$
(measured up to 60W)



HCPVT



Radiation input
 850 W/m^2
 for 7.1h / day



Overall System Yield 80%
 Energy recovery fraction from sunlight 50%

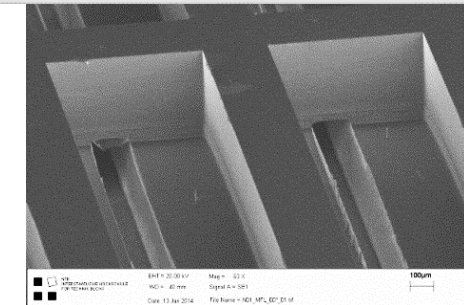
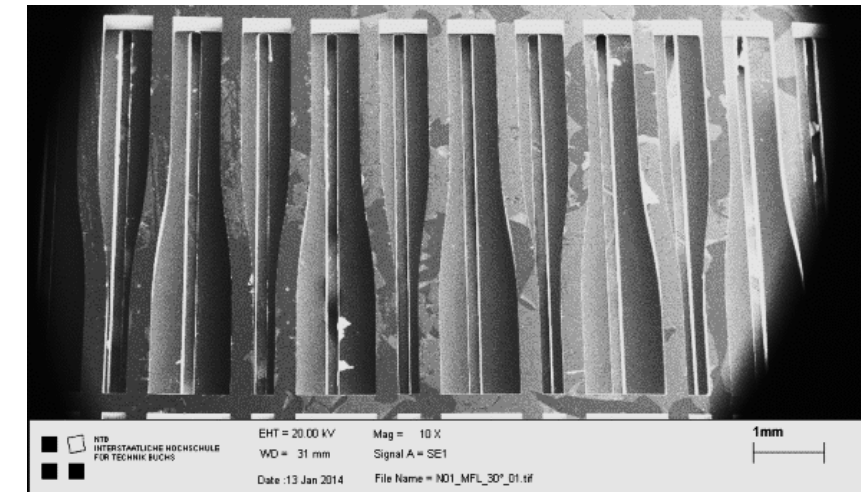
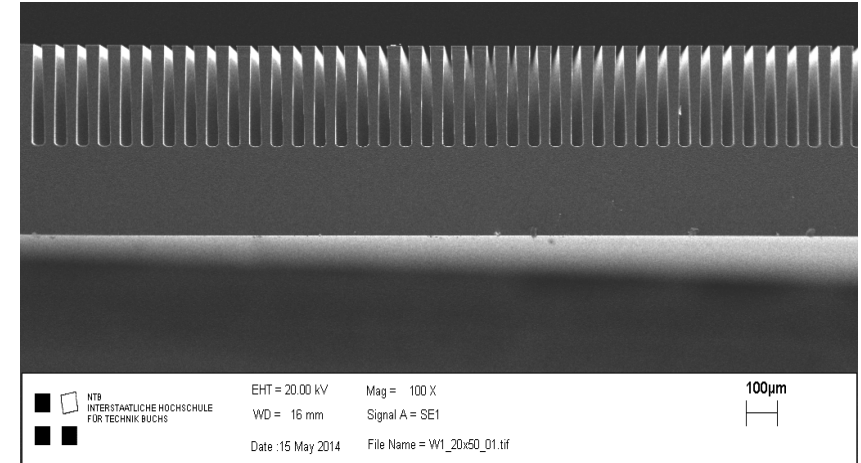
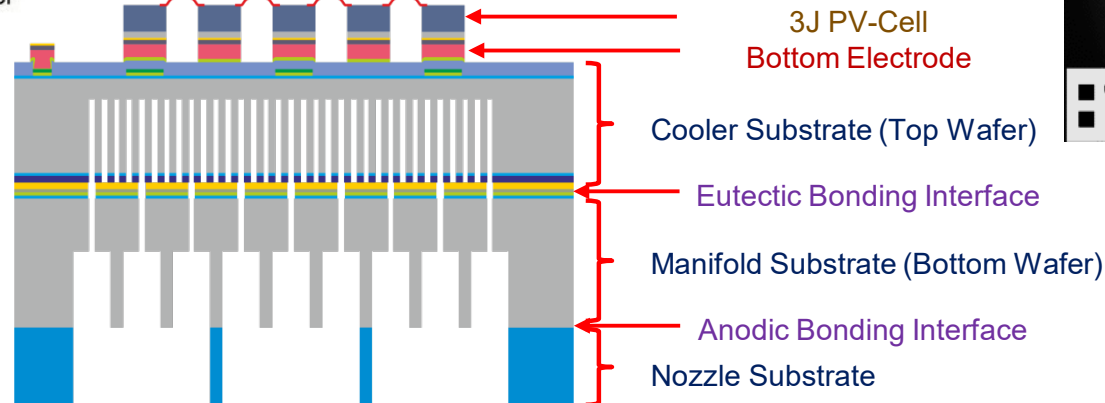
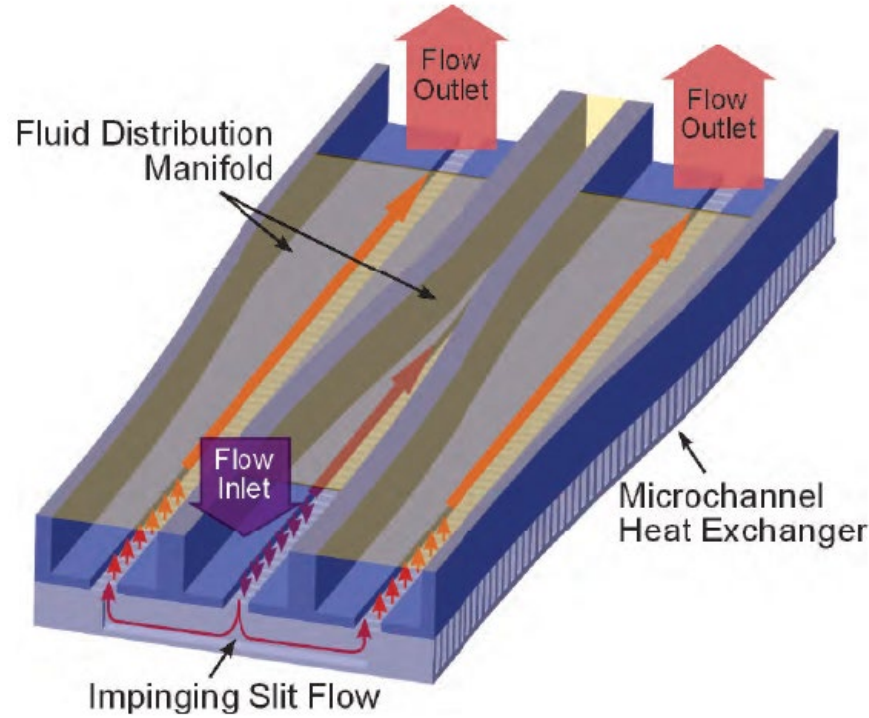
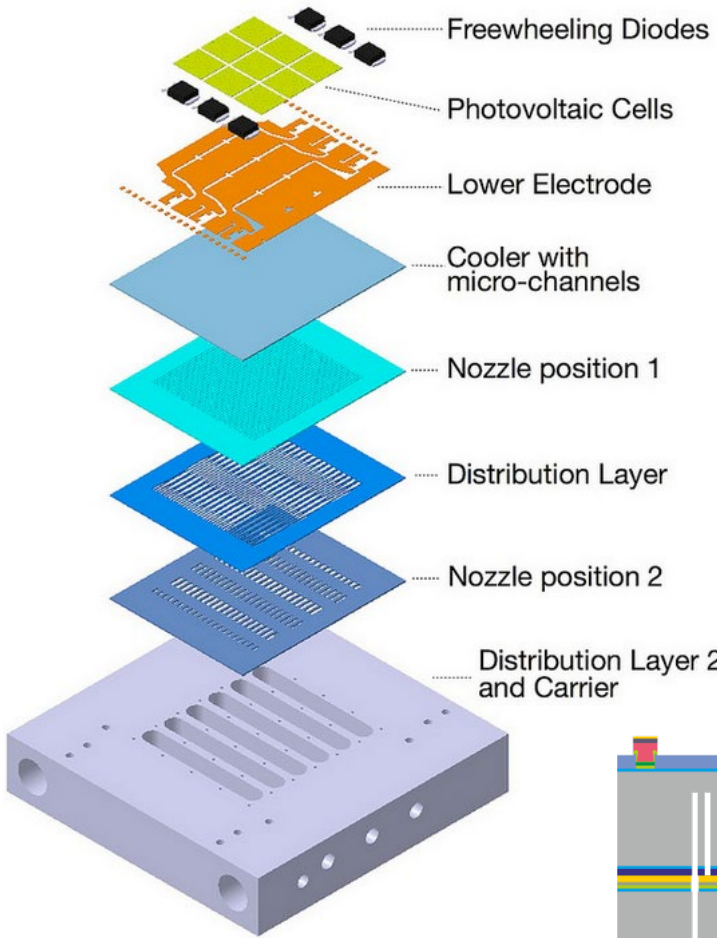
Loss in optics 20%

Receiver technology:
 Multichip package with 10x lower thermal resistance
 concentration up to 5' 000x and improved lifetime of >25 years

Electrical yield 25%
 204 W/m^2 using triple junction chip
 Potential for long-term improvement to 35%

Thermal yield: 55% 460 W/m^2 converted to:
Cooling yield of 1.3 kWh/(m² day) with a COP of 0.6
Desalination yield of 30 l/(m² day) with GOR of 7

HCPVT



Conclusion

- You have to know your **system**, its **boundaries** and **interfaces** quite well even more if the thermal load increases.
- Depending on the **amount** of heat you have **to transfer** the methods will shift towards cooling with **liquids** and or **two phase** systems even more.
- High Power System desing including cooling will be getting more crucial as the power increases and the volume decreases therefore **a wholistic** view of the system and the environment is needed.